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AVIATION AND AERONAUTICAL ENGINEERING



Above the Alps

VOLUME IX
Number 4

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THE DESIGN OF MONOCOQUE FUSELAGES
AEROMARINE MODEL 40 HULL TEST
SAFETY IN FLIGHT
THE LAIRD SWALLOW
PROPERTIES OF SPECIAL TYPES OF RADIATORS

PUBLISHED SEMI-MONTHLY

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HIGHLAND, N. Y.

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Vol. IX

September 15, 1920

No. 4

The spanning of the American continent by the extension of the New York-Orleans air mail route to San Francisco was truly one of the most important of the new era in the rapid transportation of mail. By this service the air mail will daily average 600 lb. of mail 42 hr. into San Francisco, which performance cuts the train time between the Atlantic and the Pacific coasts nearly in half. At the same time 1,800 lb. of mail will be accelerated 24 hr. into San Francisco.

The economic results of this event is bound to produce are obvious. By spending up business correspondence, the coast-to-coast air mail service indirectly stimulates production and so contributes to the welfare of the nation.

This result alone would fully justify the expense involved in the establishment of the transcontinental air mail service. But it has still another significance, perhaps more important than the first named.

The 2,600-mile airway which now stretches from New York to San Francisco constitutes probably the greatest system of regularly maintained airways in the world. From its leading fields, completely equipped with hangars, fuel and repair facilities, and situated approximately every 200 miles along the airway, will enable the mail pilots to wing their way on schedule from coast to coast. In addition the ground organization includes a series of radio stations from which weather forecasts will be sent to the mail planes through wireless telegrams. As a further safety feature most of the communities situated five miles north and five miles south of the transcontinental airway have agreed to erect in 4-foot letter the names of the cities on the west of some prominent building for the guidance of all pilots.

The Atlantic and the Pacific Coasts are thus connected by an airway ten miles broad which affords the greatest expansion that aerial navigation demands. In time of war this airway would enable our service airplanes to be dispatched in either coast under the best possible conditions. For commercial purposes its importance promises to become just as great. In fact, the performance of the air mail will undoubtedly set an example to prospective air transport lines by inspiring that of airplanes can carry mail on schedule from coast to coast they can likewise transport passengers.

The Air Mail Service is to be congratulated upon the way in which it has forged its way westward. Progressing step by step, it overcame obstacles that seemed well high insurmountable a few years ago, until it succeeded in completing the aerial link between the Atlantic and the Pacific coasts.

Nonrecurrence for Aerostatics

The experience of a standard aerostatic for aerostatics is shown at first sight. In the early days of aviation there would a considerable time in the matter of aerostatic tests, many investigators coming there were made, often using French terms some of which were a general clearly defined. The term *avulsion*, which has indifferently been used

to denote difference in incidence between the upper and lower planes and the tail plane, whereas in French it is mainly employed for expressing stagger, is a classic instance.

The instance, by the National Advisory Commission for Aeronautics, of a new and greatly enlarged aerostatic for aerostatics will therefore be warmly welcomed by the aviation world. While few changes cover in the new compilation is comparison with a previous report, the work at hand (Report No. 81) contains a great number of additional terms which the practice has confirmed in their usage. The whole compilation strikes us as being very carefully thought out, with the definitions thoroughly clear and precise. The introduction of the term *tail*, to denote "the relative distance of the wings and wing trailing on one side of the fuselage of an airplane, or between fuselages and nacelles, where there are more than one," is a happy adaptation of the French *queue* which will do a long life's work.

In view of the important position the National Advisory Commission for Aeronautics holds in the aviation world it is to be hoped that its new nomenclature will find a widespread use not only by aerostatic writers, but also by the lay press, which is altogether too prone to use nomenclature in subjects aerostatic. The worst and most commonly committed offense in this respect is the use of the term *work*—which denotes a disfigurement rather than a work, that is a very kind of aerial vehicle, be it balloon, airplane or parachute. A corrected aerial by the American press with a view to breaking the point—and their own writers—current aeronautical terms would do much good in increasing this respectable state of affairs.

Air Cooled Aero Engines

The water-cooled engine has almost monopolized the attention of American manufacturers in the evolution of aerostatic. Many advantages over the latter have been claimed and advanced as the reasons for this state of affairs, which reflects their relative popularity for aerostatic. Chief among them were greater reliability, economy, and low limitation of use of cylinders. On the other hand, lighter weight was generally awarded to the air-cooled engines. Scientific research has changed some of these arguments. It is now possible to get thermal efficiencies with air cooling quite as good, if not better, than in water-cooled engines. The construction of the fixed cylinder intake is simpler, if anything, largely on account of the elimination of the water pump. Reliability should therefore be better. Great progress in the power per cylinder and total power has also been made. The overall diameter of the larger engines is constant from the relative point of view, but there is reason to believe that research can solve this problem of keeping the head resistance within reasonable limits. All of these points hold so much promise that it is pleasant to behold the interest and support of this type shown by Army and Navy.

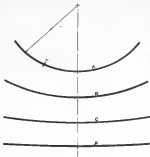


FIG. 4. ARCH OF CURVES SHOWING THE EFFECT OF CURVATURE ON THE MODULUS OF INERTIA ABOUT THE CENTERS OF GRAVITY.

$I =$ Radius of I_{xx} of any section to I_{xx} for section B

$B =$ Length of arm = 6 inches for all arcs

$r =$ Radius in inches

$I_{xx} =$ Moment of inertia about line xx through center of gravity of the arc

$r =$ Thickness of arc as in tables

$r = 2.5$

$r = 1.0$

$r = 100$

resists the ordinary long stresses. In order to compare the strengths of sections having the different curvatures shown in Fig. 4, it is then simply necessary to compare their moments of inertia about a horizontal axis through the center of gravity.

The rapid change in moment of inertia for the various sections shown in Fig. 4 is very significant in that it shows approximately what effect increasing curvature may have on the resistance to buckling. Section B, for example, has a moment of inertia about a horizontal axis through its center of gravity that is fifty times as great as that of section D. It is obvious that a change in curvature from section D to that of section B should result in a very marked improvement in resistance to buckling. That sharper curvatures give greater resistance to buckling is borne out by the tests, as will be seen from the results in Table I.

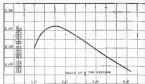


FIG. 5. COMPARISON OF STIFFNESS MODELS BY ELLIPSE AND CIRCLES.

Major axis b of ellipse constant = 1.6

Major axis a of ellipse variable

Section models taken about the minor axis

Thickness t of ellipse = circumference of circle c = thickness of shell; $a = 0.5$ for all sections

Assuming the same fiber stress, it remains to be seen which type of complete section gives the greater resisting moment for a given weight. A comparison of strength is now made between forewings such as having an elliptical section, and those having a circular section. For this purpose an elliptical section having a minor axis of 3-foot length was assumed and the ratio of the length of the major axis to the minor axis was varied, leaving the area of the section of the shell c , or the weight of the forewing constant in each case. Circular sections corresponding to each of the ellipse sections were determined such that the circumference of the circle is equal to the perimeter of the corresponding ellipse and the thickness of the circular shell is equal to the thickness of the ellipse shell. Moments of inertia about the major axis were computed in each case and the section modulus determined for each. The same was done for the circular sections. The results of these computations are given in Fig. 4, in which the ratio of the section modulus of the ellipse sections to the section modulus of the circular section is plotted against the ratio of the length to the minor axis of the ellipse. Since the external bending moment is practically constant for a given stress it is directly proportional to the section modulus, the forewing section having the greatest section modulus will resist the applied bending moment. On the basis of their section moduli, Fig. 5 shows that for ratios of major and minor axes from 1 to about 3, the ellipse section has a slight advantage over the circular section. Add to this the very decided advantage of greater resistance to buckling on account of the sharper curvature at the points most highly stressed, it will be seen that the ellipse section should show appreciably superior strength over the circular section.

The ellipse section also provides greater depth for the pilot or passenger.

Aeromarine Model 40 Hull Test

The tests were made in accordance with instructions received from the Bureau of Construction and Repair and are recorded in the appendix of this report.

To simulate flying condition the hull was suspended by means of the flying cables attached to the hull and the whole arrangement constructed so that the hull would be balanced at the center of pressure for low speed.

In making the test for flying condition the hull was suspended in the same manner and the test engine was attached twenty pounds per square foot, the part of the hull aft of the rear flying wire acting as a counterweight.

The test for landing load was made with the hull supported on a steel landing track, and loads added in increments of a unit landing load.

Date and Place

The tests were made at the Aeromarine Plant and Motor Company's factory, Kynor, New Jersey. Mr. Fred Kitch of the Engineering Department of the Company was in charge of the tests. Eugene Edmund A. Whiting, U. S. N. R. F., represented Lieutenant A. Rhoads, Inspector of Naval Aircraft, U. S. N.

Mr. R. H. Buel of the Bureau of Construction and Repair was present at the first two tests. The tests were made on the following dates:

I. Flying Condition—July 10, 1916.

II. Flying Condition—July 10, 1916.

III. Landing Condition—July 13, 1916.

Procedure

Test No. 1, Flying Condition—Hull was suspended from staging by means of the main lift cables, and simulated flight conditions to be at low speed with the center of pressure at 30 per cent aft of the leading edge of the main wing chord. The test loads were computed at the weight of the following parts: Hull, fuel, pilot, passenger, controls, instruments, etc. A tail load was added to balance the machine and induce a flexure in the lift wires corresponding to the center of pressure location specified. The loads were applied by means of steel lines and distributed over the hull considering as nearly as possible the distributed load weight. The hull in this condition was tested to a factor of safety of six.

Test No. 2, Flying Condition—For this test the hull aft of the main wing chord was a counterweight with a load of 20 pounds per square foot on the tail surface. The same set up was used as in Test No. 1. The hull was kept in balance with center of pressure at 50 per cent of chord by loading the fore part of hull. The hull successfully carried a load of 20 pounds per square foot, on the tail surface, acting as a counterweight.

Test No. 3, Landing Condition—The hull was supported on a reduced landing track, the area of the track simulating the position the hull takes at the beginning of a steep landing. The engine section was included in the set up. The load imposed included all the weights act contained in the hull and tail, i. e., wings, wing floats, engine, oil, tanks, radiator and

motor, etc. The load was carried by the engine section trim-up. The structure was tested as follows:

A factor of safety of twelve for load representing the wings, wing floats, etc. A factor of safety of ten for load representing the engine, oil, tanks, radiator and motor, etc. A factor of safety of ten for loading representing one quarter the weight of the upper wings, motor, etc., acting in a horizontal direction and assumed as a thrust.

Results

In tests Nos. 1 and 2, the loads were applied until a factor of safety of six was obtained. The hull successfully stood the test.

In test No. 3, a factor of safety of ten was obtained for all parts. At this point it was not possible to continue adding the horizontal load and the engine loads. The loading, however, was added to the engine section (used until a factor of safety of twelve was obtained without signs of failure).

Construction of Hull

The detailed construction of the Aeromarine Model 40 Flying Boat hull shows no radical departure from the current practice.

The hull consists of back-up transverse frames of spruce spaced to shape with the bottom ribs extending past the side ribs, the ribs of which are held in the side ribs by one-half inch to three-quarter inch struts. This construction forms the first on the forward part of the hull. All frame joints are joined with glue and held in place by 2 plywood gaskets on each side of a frame at each corner, riveted with copper rivets and nuts. The longitudinal transverse members of oak skin are riveted to receive the plating, spruce deck changes of the same type, oak or oak forward and bottom, spruce after bottom with two auxiliary keelsons or stringers, equally spaced. The chines and deck members overlap at the stop rib and are well riveted to take the stress at this point. The sides and top are planked with three ply birch and poplar veneer, fastened with brass screws. The bottom is planked with teakwood, laid up of Port Orford cedar with a same layer of cotton fabric with marine glue and quilted with barbed wire keels. The landing struts are taken by two heavy spruce thrust braces leading from the forward wing beam to the bottom of the hull structure.

The hull has four water tight bulkheads with drain plugs readily accessible and one strengthening bulkhead. The reinforcement provided for two people and the control engine, contained in the hull, is made of Port Orford cedar with a same layer of cotton fabric with marine glue and quilted with barbed wire keels. The landing struts are taken by two heavy spruce thrust braces leading from the forward wing beam to the bottom of the hull structure.

Flying Condition Test

The hull was suspended from suitable staging by means of the main lift cables. The upper part of the staging was which

TABLE I COMPARISON OF STIFFNESS MODELS BY ELLIPSE AND CIRCLES. ALL IN THE SAME WEIGHT. Ratio of Moment of Inertia About the Centres of Gravity to that of the Circle Model.									
Type of Section	Ratio of Moment of Inertia to that of the Circle Model		Ratio of Area to that of the Circle Model		Ratio of Moment of Inertia to that of the Circle Model		Ratio of Area to that of the Circle Model		Ratio of Moment of Inertia to that of the Circle Model
	Ratio of Moment of Inertia to that of the Circle Model	Ratio of Area to that of the Circle Model	Ratio of Moment of Inertia to that of the Circle Model	Ratio of Area to that of the Circle Model	Ratio of Moment of Inertia to that of the Circle Model	Ratio of Area to that of the Circle Model	Ratio of Moment of Inertia to that of the Circle Model	Ratio of Area to that of the Circle Model	
Elliptical	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Parabolic	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Triangular	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Quadrilateral	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Pentagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Hexagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Heptagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Octagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Nonagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Decagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Undecagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Dodecagonal	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Ratio of strength of two sections of equal weight	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00



SET-UP FOR FLYING CONDITION TEST

SET-UP FOR LANDING CONDITION TEST

A New High Performance Wing

A novel form of airplane wing, which appears to go a long way toward improving the economy of commercial flight, has just been introduced in England under the name of Alaka wing. The aerodynamic data and the outline of the new wing are given below. Although in its present form the wing needs a great improvement on the ordinary needed for load carrying at moderate speeds, it does not, as the opinion of the inventor, represent the maximum attainable.

The sharp characteristics of the Alaka wing are a straight trailing edge, a negative dihedral leading edge, a deep camber, and a washout in chord, together and sometimes toward the tip. In addition to wide camber, several tests and experiments have been made for some time by the technical staff of the Blackburn Aeroplane and Motor Co., of Leeds, in order to determine the agreement between model and full scale work. These trials are still in progress, but it is stated that very satisfactory results are being obtained.

Angle of Incidence (Degrees)	$K_d \times 10^3$	L/D	Quality of Pressure Coefficient
0	0.0	15.0	0.000
2	0.1	14.5	0.001
4	0.2	14.0	0.002
6	0.3	13.5	0.003
8	0.4	13.0	0.004
10	0.5	12.5	0.005
12	0.6	12.0	0.006
14	0.7	11.5	0.007
16	0.8	11.0	0.008
18	0.9	10.5	0.009
20	1.0	10.0	0.010
22	1.1	9.5	0.011
24	1.2	9.0	0.012
26	1.3	8.5	0.013
28	1.4	8.0	0.014
30	1.5	7.5	0.015
32	1.6	7.0	0.016
34	1.7	6.5	0.017
36	1.8	6.0	0.018
38	1.9	5.5	0.019
40	2.0	5.0	0.020
42	2.1	4.5	0.021
44	2.2	4.0	0.022
46	2.3	3.5	0.023
48	2.4	3.0	0.024
50	2.5	2.5	0.025
52	2.6	2.0	0.026
54	2.7	1.5	0.027
56	2.8	1.0	0.028
58	2.9	0.5	0.029
60	3.0	0.0	0.030

A reference to the accompanying table will show that not only is the maximum lift coefficient unusually high, but the maximum L/D and the lift coefficient corresponding to maximum L/D are also extremely good. The maximum lift coefficient is 32.2 and occurs at an angle of incidence of 35 deg. As the maximum lift coefficient of the ordinary wing is somewhere between 25 and 30 it will be very clear, for the same load and speed, the wing area can be reduced to about three-fourths that necessary with the ordinary wing. This is not the only, nor perhaps the greatest, advantage of the Alaka wing, as it is called by the designer, the maximum lift coefficient with a correction still has to be added 10% as high as 32.0, which compares favorably with the orthodox wing, and the lift coefficient corresponding to the maximum L/D is 28.8, or, on the maximum lift coefficient of the ordinary wing. This, it will be seen makes for economy of flight, especially in machines carrying a high load at moderate speeds.

The "Folium" 6-Eng. Lorry

An interesting example of a recuperating design using Alaka wings is the machine shown here. The wings and fuselage are both constructed of anodized glazing on the principle used in modern five-bay boats, thus simplifying the machine and eliminating the bulk of the metal parts, fabric, etc., and will give great durability, a feature of the first importance in relation to recuperation charges.

The machine is a non-flier, being designed to dispense with all the usual elements.

The crew consists of a pilot seated right forward and a navigator in charge of the engine-room, all. The engine is a two-cylinder, Napier-Lucas, and the machine is designed to fly and climb even if one engine breaks down. Reliability of service is assured by this power to fly on one engine and by the superabundance of the engine in flight, coupled with the fact that all work is normally done at half-power.

The design has been prepared on a most conservative basis in all respects. For instance, an allowance of 5% per cent on all structural weight above the calculated figure has been included, and the factor of safety has been taken as 1.5 in the calculations, whereas 3 is a more realistic mark for the cargo

machines, further, a 20-m.p.h. head wind is assumed in calculating the weight of fuel carried.

Specifications		140 ft.	140 ft.
Length	140 ft.	140 ft.	140 ft.
Height	140 ft.	140 ft.	140 ft.
Span	140 ft.	140 ft.	140 ft.
Wing Area	140 ft.	140 ft.	140 ft.
Weight	140 ft.	140 ft.	140 ft.
Thrust	140 ft.	140 ft.	140 ft.
Speed	140 ft.	140 ft.	140 ft.
Altitude	140 ft.	140 ft.	140 ft.
Endurance	140 ft.	140 ft.	140 ft.
Range	140 ft.	140 ft.	140 ft.
Power (4000 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (1000 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (500 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (250 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (100 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (50 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (25 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (10 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (5 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (2 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (1 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.5 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.2 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.1 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.05 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.02 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.01 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
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Power (0.000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.00000000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.000000000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000000000005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000000000002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0000000000000000000000000000000000000001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0002 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.0001 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.005 m.p.h.)	140 ft.	140 ft.	140 ft.
Power (0.002 m.p.h.)	140 ft.	140 ft.	14

The weights of the radiator and the water contained in them were obtained from geometrical considerations and the densities of copper and water, assuming the Laporte type of construction. The weight is given by the equation

$$W = 0.0507 \times \pi \times \quad (4)$$

where W = weight of pipe and contained water, in pounds per square foot frontal area.

A left-to-right ratio of 5:4 was assumed for the airplane, and the horsepower absorbed (per square foot) is

$$H.P. = \frac{K + \frac{W}{P}}{5.4} \quad (5)$$

Figure of merit is the ratio of the power dissipated to the power absorbed.

$$F.M. = \frac{Q}{H.P.} \quad (6)$$

The following example illustrates the use of the equations. Let it be required to obtain the figure of merit of a radiator with plates 1.0 in. thick, $\frac{1}{2}$ in. pitch, 16 in. deep, and at 120 m.p.h.

In equation (3) for mass flow of air,

$$M = 0.116 \sqrt{\frac{P-1}{P}} \left(F \left(1 - e^{-0.011 \sqrt{\frac{P-1}{P}}} \right) \right)$$

$$P = \text{pitch} = 0.5$$

$$F = (\text{face area}) = 1.6 \times 0.6025$$

$$F = \text{depth} = 16$$

$$F = \text{speed} = 120$$

$$M = 0.116 \sqrt{\frac{0.5}{0.5}} (1.6 \times 0.6025) \left(1 - e^{-0.011 \sqrt{\frac{0.5}{0.5}}} \right)$$

$$M = 0.116 \times 0.975 (1.6 \times 0.6025) (1 - e^{-0.011})$$

$$M = 12.55 (0.975) = 12.55 \text{ lb. per sq. ft. per sec.}$$

In equations (5) for energy dissipated,

$$Q = 34.6 M \left(1 - e^{-0.0055} \right)$$

$$A = 0.20$$

$$B = 0.0055$$

$$Q = 34.6 (12.55) \left[1 - e^{-0.0055 (12.55)} \right]$$

$$Q = 350 (1 - e^{-0.0695}) = 350 (0.0695)$$

$$Q = 77.3 \text{ H.P. per sq. ft. per } 100^{\circ} \text{ F.}$$

In equations (3) for heat resistance,

$$K = 24 \left(0.0016 \pi + 0.00000025 \right),$$

$$\pi = 3.14$$

$$K = 24 (120)^2 \left(\frac{0.0016 \pi}{3.14} + 0.00000025 (16) \right)$$

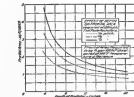


FIG. 5

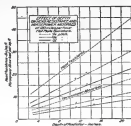


FIG. 4

$$K = 24 (120)^2 (0.000025 + 0.0000025)$$

$$K = 17.27 \text{ H.P. per sq. ft.}$$

From equation (4),

$$W = 0.0507 (24) (16) = 21.6 \text{ lb. per sq. ft.}$$

From equation (2),

$$H.P. = \left(K + \frac{W}{P} \right) \left(\frac{F}{375} \right)$$

$$H.P. = \left(17.27 + \frac{21.6}{0.5} \right) \left(\frac{1.6}{375} \right) = 0.30 \text{ H.P. per sq. ft.}$$

From equation (6),

$$F.M. = \frac{Q}{H.P.} = \frac{77.3}{0.30} = 254$$

Description of Curves

Plot 1 shows figure of merit computed with the aid of the above equations for various depths and for speeds of 60, 90,

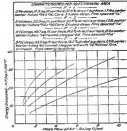


FIG. 6

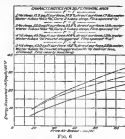


FIG. 6

and 120 m.p.h., and for spacings of $\frac{1}{2}$, $\frac{3}{4}$, and 1 in. between plates. The curves illustrate the following points:

- (1) The $\frac{1}{2}$ in. pitch gives, in general, a higher figure of merit than those of closer spacings.
- (2) For high speeds the radiator may be somewhat deeper than for low speeds.
- (3) For the higher speeds the most efficient depth is considerably greater than those in common use with the radiator type of core.
- (4) For the higher speeds, the figure of merit is practically at its maximum value over a considerable range of depth, so that if considerations of construction make it desirable to spread the frontal area to a maximum, a reasonable increase in depth beyond the optimum will have but a small effect on the figure of merit.

The effect of depth on the properties of the radiator is shown

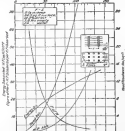


FIG. 7

in plots 5, 6, and 7. The values of "area required per 100 hp." in plot 5 are 100 times the reciprocals of the corresponding values of energy dissipated (in hp./sq. ft.) of plot 2.

The following example illustrates one way in which the curves may be used. For a radiator with $\frac{1}{2}$ in. pitch, to be used at 120 m.p.h., the maximum figure of merit is given on plot 1 as 11.7 at 16 in. depth. From plot 6 the frontal area of radiator required with 16 in. depth is 1.55 sq. ft. per 100 hp. If, in order to reduce the frontal area, a depth of 20 in. should be used, the area required would be 1.50 sq. ft. per 100 hp. (from plot 6), and the figure of merit would be 11.2 (from plot 1). The horsepower absorbed would be increased from 5.5 to 6.1 per sq. ft. (plot 7). Since, however, the frontal

area may be reduced in the ratio $\frac{1.50}{1.55}$ the actual power absorbed that should be compared with the value for 16 in. depth would be $(6.1) \times 0.965$. These results may be summarized as follows:

	16 in.	20 in.	Change
Area required, sq. ft./100 hp.	1.55	1.50	-3.2%
Horsepower absorbed, per sq. ft. when required at 16 inches	5.5	5.70	+3.6%
Figure of merit	11.7	11.20	-4.2%

A careful distinction should be made between radiators whose water tubes are smooth, flat plates and other types using perforated plates, or deep and narrow tubes placed in rows, as shown in Fig. 8.

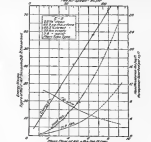


FIG. 8

one behind the other. Even in the water tubes, or spaces between them in the direction of the air flow, some very great increase in heat resistance and decrease in mass flow of air, and although the heat transfer per square foot of cooling surface may be increased by the great turbulence caused, it is at a very heavy cost in heat resistance, and with a decrease in figure of merit.

The effect of holes in the water tubes, and of spaces between them in the direction of the air flow is taken up later in this report under "Properties of Whirling Radiators," in which the properties of six such types are given.

Properties of Fin and Tube Radiators

Radiators of the "fin and tube" type are characterized by high heat resistance, and low heat transfer at high speeds. With the possible exception of certain elongated types that use in the wing they are unsuitable for airplane use.

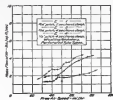


FIG. 9

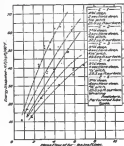


FIG. 10

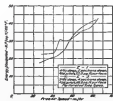


FIG. 11

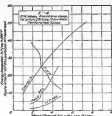


FIG. 12

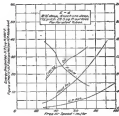


FIG. 13

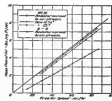


FIG. 14

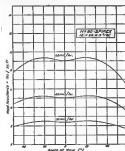


FIG. 15

The properties of "fin and tube" types of cooling radiators are definitely different from those of the better radiator types and warrant special mention.

In general, the fin and tube types are characterized by high head resistance and by low heat transfer at high speeds. The low heat transfer is accounted for by the small flow of air through the radiator and by the large amount of radiant cooling surface. For radiators as ordinarily made with depths of 6 in. or less, head resistance has been shown to be due principally to the impact of the air on the front face and surface on the rear, and only to a small degree to skin friction on the walls of the air passages. With the fin and tube construction the effects of impact and surface are exaggerated; for each separate water tube is subjected to impact on one side and surface on the other, with the result that the total (projected) area subjected to impact and surface—on all tubes—is much greater than that associated in a radiator of the same size but of solid construction. To the effect of the impact and surface must be added the effect of skin friction on the fins.

Excessive head resistance, accompanied by low heat transfer, makes the fin and tube types unsuitable for use on an airplane where they would be exposed to a constant flow of air at a high speed. The same compact types, however—notably F-4, which has large water tubes with crimped spiral fins nearly touching each other—show a relatively high rate of heat transfer at very low speeds, which enables them to operate as air transfer and lower speed antiskid. Indeed the type F-4 must definitely dissipate a considerable amount of heat with correction currents only.

The accompanying curves show the heat transfer (rating decreased) for five types of core, in terms of the mass of air flowing through the core. The heat transfer is expressed in horsepower per square foot of frontal area, for a difference of 100 deg. F. between the mean temperature of the water and the temperature of the entering air. Mass flow of air is expressed in pounds per second per square foot of frontal area. Energy dissipated is also shown in terms of free air speed; that is, the speed of the mass in which the radiator is immersed. Plot 3 shows also the figures of merit of the radiator, which is the ratio of the horsepower dissipated to the horsepower absorbed.

The attempt to understate the relation between the mass flow of air through the core and free air speed was unnecessary.

but is the case of the type F-4 (with spiral fins); for the air flow was too small to be measured with the instrument used on the other radiators.

In general, it may be stated that fin and tube radiators are unsuitable for airplane use, with the possible exception of a type similar to F-4, placed on the wing, where the mass flow of air must be very small (even less than for the nose position), and consequently head resistance is not necessarily a detriment.

Properties of Whisking Radiators

The construction of some types of radiator is such that at certain air speeds they produce a whisking sound. These so-called "whisking radiators" are characterized by the following points:

- (1) Unusual conditions of air flow, resulting in irregularities in the relation between different portions. For example, mass flow of air through the radiator is not proportional to free air speed as in ordinary radiators, and head resistance is not proportional to the square of free air speed.
- (2) High heat transfer for a given mass flow of air through the radiator.
- (3) Very low flow of air through the radiator for a given free air speed.
- (4) In many cases, low heat transfer for a given free air speed.
- (5) Very high head resistance and horsepower absorbed.
- (6) Low figure of merit.

If solid tubes made of smooth flat plates, continuous from front to rear of the radiator, are substituted for the rows of tubes of the whisking radiators described, the figure of merit will be greatly increased.

Certain types of cooling radiators that while in an air stream show peculiar properties, and while radiators of this construction are not recommended, they are being used in some cases, and appear to be worthy of special mention.

Types of Core Tested

The whisking radiators tested at the Bureau of Standards fall into two general classes:

- (1) Plain water tubes, and
- (2) Perforated water tubes.

The photographs show the forms of construction. In each case the radiator is made up of separate water tubes about 3 in. deep (on the diameter of air flow), arranged in rows. In addition, the plain tube type has flat spaced 2 in. apart.

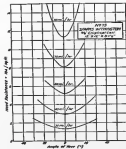


FIG. 16

The test sections included one of the plain tube type, designated as E-6, and five perforated tube types, made up in different depths, and with different spacings between rows of tubes.

These sections are designated as follows:
E-3, 5-16-in. pitch, 2 tubes deep.
E-3, 5-16-in. pitch, 3 tubes deep.
E-3, 5-16-in. pitch, 4 tubes deep.
E-4, 1/2-in. pitch, 4 tubes deep.
E-6, 7/8-in. pitch, 4 tubes deep.

Cause and Effect of the Whistles

The form of construction leaves continuous air passages across the radiator, that is, perpendicular to the direction of the air stream. In the plain tube type these air passages occur between the water tubes, and in the perforated tube types not only between the tubes but at each perforation. All through the radiator there are short or long air sacs, across the ends of which air is blowing, with the result that vibrations are set up in the short columns and perpendicular to the air stream. The resulting whistle will of course vary widely in intensity and in pitch as the speed of the air stream varies, and conditions of resonance have very marked effects, not only upon the sound, but upon the properties of the radiator.

By the vibration of the mass columns, air is alternately being forced into and withdrawn from the fast moving stream. Air drawn out of the stream will be retarded and air forced into it will be accelerated, thus acting as a drag on the stream. These two effects cause a great decrease in the flow of air through the radiator and a great increase in heat resistance. At the same time the very great turbulence caused in the air stream results in a high heat transfer per square foot of cooling surface for a given mass flow of air through the radiator, and this increase in heat transfer may be so great as to counterbalance the decrease in R.T. flow but is not great enough in any case observed to overcome the disadvantage of the increased heat resistance.

DESCRIPTION OF CURVES

The accompanying curves show the properties of the six types of radiator, expressed as follows (all values have been reduced to an air density of 0.0750 lb./cu. ft.):

Free air speed, in miles per hour.
Mass flow of air through the radiator, in pounds per second per square foot frontal area.

Energy dissipated, in horsepower per square foot per 100 deg. Fahr. difference between the temperatures of the entering air and the mean of the temperatures of the radiator and leaving water.

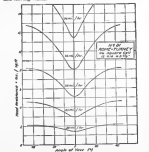


FIG. 17

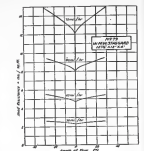


FIG. 18

Heat resistance, in pounds per square foot frontal area.
Horsepower absorbed, in horsepower per square foot frontal area.

Figure of merit is the ratio of the horsepower dissipated to the horsepower absorbed.

Plot 1 shows the properties of the plain tube type, E-6. Its heat transfer is high, but its heat resistance is also high, and the figure of merit is low. The great weight of the radiator accounts for its very low figures of merit at low speeds.

The relation between the mass flow of air through the radiator and free air speed is shown for three of the perforated tube types in plot 2. For ordinary radiators this relation is linear and it is possible as for the plain tube whistling type. The irregular form of these curves shows clearly the effect of the peculiar conditions of air flow in the radiator, the sudden changes in slope of the curve corresponding to sudden changes in mass of the whistle.

Plot 3 shows energy dissipated (heat transfer) in terms of mass flow of air. The curve for the type E-4 was determined by interpolation,* since the water losses had been removed in order to measure its heat resistance. Too much importance should not be assigned to the fact that these three curves show a high heat transfer with a given mass flow of air. For Plot 4 shows that the highest curve of Plot 3—that for E-3, four tubes deep, 5-16-in. pitch—becomes low when heat transfer is plotted against free air speed.

Plots 5 and 6 show the complete properties of the type E-4, in terms of mass flow of air, and in terms of free air speed, respectively. The curves are reliable within the ranges noted, but the irregular relation between mass flow of air and free air speed makes interpolation extremely doubtful.

Heat resistance of ordinary radiators is nearly proportional to the square of free air speed, but the interpolation in the heat resistance curves of Plots 1 and 6 shows that this is not the case with the whistling types.

COMPARISON BETWEEN WHISTLING RADIATORS AND PLAIN PLATE RADIATORS.

The foregoing statements should not be interpreted as applying in any degree to radiators whose water tubes are flat plates with square surfaces, and continuous from front to end of the radiator.

* Type E-3, E-4, and E-6 are each four tubes deep, and are therefore identical in heat, and therefore heat loss, respectively. The heat transfer for E-3 and E-6 is given in the figures of merit, and is not the heat loss of front, and this proportionality was used in interpolating for E-4.

The whistling types are characterized by low air flow and often low heat transfer (for a given free air speed), by high heat resistance, and by a low figure of merit, but the flat plate types are characterized by high air flow and heat transfer (for a given free air speed), by low heat resistance, and by a high figure of merit. The following comparison tables show the superiority of the flat plate types over the whistling types.

TABLE I
PERFORATED TUBE TYPE AND FLAT PLATE TYPE OF THE SAME PRICE AND PRACTICALLY THE SAME DESIGN.

Type	E-4 perforated tube type	E-6 flat plate type
Pitch, inches	5.16	7.87
Depth, inches	5.16	5.16
Number tubes, square feet, per square foot frontal area	59.4	59.4
Mass flow of air, pounds per second per square foot	0.150	1.40
Heat resistance, in pounds per square foot	1.00	0.10
Heat transfer, in horsepower per square foot	0.04	0.13
Figure of merit	0.04	1.30

Test data are not available for a direct comparison of plain tube and flat plate types of radiator of the same pitch, but the plain tube whistling type E-6 of three-fourths inch pitch may be compared with flat plate types of one-half inch pitch. It was previously shown that for flat plate types a pitch of one-half inch is better at high speeds than the closer spacings, and there seems to be no reason to suppose that a pitch of three-fourths inch would be less efficient. A comparison of the two types would appear to give an advantage to the use of three-fourths inch pitch. Two depths of the flat plate type are



FIG. 19

standardized, giving approximately the same direct cooling surface not including fins, and the same actual depth of ribs, respectively, as the whistling types. A plain tube whistling type (E-3, described in a French report, is also included.

TABLE II
COMPARISON OF FLAT PLATE AND PLAIN TUBE RADIATORS
Free air speed, 150 miles per hour

Type	Flat plate type	Plain tube type	E-6 flat plate type
Pitch, inches	5.16	5.16	7.87
Depth, inches	5.16	5.16	5.16
Number tubes, square feet, per square foot frontal area	59.4	59.4	59.4
Mass flow of air, pounds per second per square foot	0.150	0.150	1.40
Heat resistance, in pounds per square foot	1.00	1.00	0.10
Heat transfer, in horsepower per square foot	0.04	0.04	0.13
Figure of merit	0.04	0.04	1.30

Free air speed, 50 miles per hour.

Type	Flat plate type	Plain tube type	E-6 flat plate type
Pitch, inches	5.16	5.16	7.87
Depth, inches	5.16	5.16	5.16
Number tubes, square feet, per square foot frontal area	59.4	59.4	59.4
Mass flow of air, pounds per second per square foot	0.150	0.150	1.40
Heat resistance, in pounds per square foot	1.00	1.00	0.10
Heat transfer, in horsepower per square foot	0.04	0.04	0.13
Figure of merit	0.04	0.04	1.30

CONCLUSIONS

The above tables serve to show clearly that any increase in heat transfer caused by perforations in the water tubes or spaces between them is the decrease of air flow as in the case of a great increase in heat resistance and is accompanied by a decrease in the figure of merit. The same result has been found in the case of turbulence caused in radiator columns, and indeed no type of radiator is known to be known in which an artificial increase of turbulence is not accompanied by a decrease in figure of merit. For use in obstructed positions, such as the nose of the fuselage, it may



FIG. 20

be necessary to sacrifice figure of merit of the radiator case for the benefit of heat transfer, but for unobstructed positions it appears that smooth straight air passages through the radiator should be provided.

EFFECTS OF YAWING AIRPLANE RADIATORS

Different radiators show different effects on being inclined at an angle to the direction of the air stream, but in general the results (for angles up to 30 deg.) are as follows:

- (1) Decrease in air flow through the case.
- (2) Increase in heat resistance.
- (3) In some cases slight increase in the heat transfer, for angles up to 30 deg. or 35 deg.

Wind-tunnel tests on radiators for streamlined engines have usually been made with the face of the radiator normal to the direction of the wind, so that although local eddies might be set up, the general direction of the air stream was straight as it passed through the radiator. If the aim of the air passages are inclined at an angle with the general direction of the air stream, or, in other words, if the face of the radiator is not normal to the air stream, the properties of the radiator will be somewhat changed, and it is the purpose of this report to show the general flow of these changes.

The effect of turning the radiator at an angle with the air stream, or turning the radiator, is of interest in connection with the following conditions:

- (1) Radiator mounted in the propeller slip stream, where the air strikes the radiator at other angles than normal to its face.
- (2) Radiator required in any position (such as in the wing) where the axis of its passages for the air are not parallel to the direction of motion of the plane.



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